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Engineering Analysis with Boundary Elements

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Revisit of two classical elasticity problems by using the Trefftz method

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ARTICLE INFO

Article history: Received 6 September 2008 Accepted 9 December 2008 Available online 30 January 2009

Keywords: Trefftz method Elasticity Lamé problem Stress concentration factor Semi-inverse method

ABSTRACT

In this paper, the two classical elasticity cases, Lamé problem and stress concentration factor (SCF), are revisited by using the Trefftz method instead of the inverse or semi-inverse approach in the previous study. First, the Timoshenko and Goodier's approach is reviewed. Based on the superposition principle and the concept of taking free body, the problem of stress concentration factor as well as Lamé problem can be solved without any difficulty in a direct way using the Trefftz method.

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1. Introduction

For solving boundary value problems of elasticity, it is usually difficult to find an analytical solution which satisfies the partial differential equations and given boundary conditions at the same time. In some cases, the inverse method or the semi-inverse method can be used. In the inverse method, a solution is found in priori such that it satisfies the governing equation and boundary condition. We can obtain the solution through this way for a luck cases, but it is not a logical way. For the semi-inverse method, certain assumptions about the components of displacement strain are made at the beginning. Then, the solution is confined by satisfying the equations of equilibrium and the boundary conditions. In the classical elasticity, the semi-inverse method was often employed to derive the analytical solution for simple problems. For example, the Saint-Venant torsion solution is a typical case which was obtained by using for the semi-inverse approach.

The Trefftz method was first presented by Trefftz in 1926 [1]. It can be seen as one kind of boundary-type methods. Until now, there are many Trefftz methods developed including direct and indirect formulations. The basic concept of Trefftz method is superimposing the *T*-complete functions which satisfy the governing equation, and the unknown coefficients are determined by matching the boundary conditions. The *T*-complete functions for plane elasticity problems have been already presented by some researchers [2,3]. The solution procedure is easier than other boundary-type methods, e.g. the boundary element method.

Moreover, the formulation of the Trefftz method is regular and singular integrals are not required to calculate. Therefore, many applications for the Laplace equation [4], the Helmholtz equation [5], the Navier equation [6] and biharmonic equation [7] were done. More applications were summarized in Refs. [8-10]. Recently, Chen et al. [11] linked the two methods, Trefftz method and method of fundamental solutions, through the degenerate kernel for Laplace and biharmonic equations. They also found that all the Trefftz bases are imbedded in the degenerate kernel for the fundamental solution. Later, Schaback [12] also presented an article to discuss on this issue. Kaw et al. [13] used the finite element method to study the problem and compared with the analytical and experiment results. It is very useful to help students to study and understand the problem. Chen et al. [14] used the null-field boundary integral method to revisit the two classical elasticity problems, Lamé problem and the problem of stress concentration factor (SCF). However, we do not find that problems have been solved by using the Trefftz method in the literature to our best knowledge. Moreover, Timoshenko and Goodier's directly used $cos(2\theta)$ in their book. Maybe the readers do not know the reason why only $cos(2\theta)$ is chosen. Based on the Trefftz method, we provided another viewpoint to derive solutions. Therefore, we will attempt to revisit the two classical elasticity problems by using the Trefftz method.

In this paper, we employ the Trefftz method to deal with the Lamé problem and the problem of stress concentration factor. This approach is seen as an analytical method and the solution is derived in a natural and logical way, once the Trefftz base and its coefficient can be determined. For the two problems, they will be revisited by using the Trefftz method. Therefore, a direct way of solution for elasticity problems is our goal.

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2. Methods of solution

2.1. Problem statements

The two classical problems in the Timoshenko and Goodier's book [15] are revisited. One is an infinite plate with a circular hole subject to remote tension (stress concentration factor problem) and another is an annular cylinder subject to uniform pressures (Lamé problem), as shown in Figs. 1 and 2, respectively. The medium is considered as an isotropic, elastic and homogenous body. The governing equation is

$$(\lambda + G)\nabla(\nabla \cdot u(x)) + G\nabla^2 u(x) = 0, \quad x \in \Omega,$$
 (1)

where u(x) is the displacement, Ω is the domain of interest, ∇^2 is the Laplacian operator, and λ and G are the Lamé constants for the isotropic elasticity.

2.2. Review of Timoshenko and Goodier's solution

2.2.1. Lamé problem

This problem was first solved by Lamé [16]. According to the axial symmetry property, Timoshenko and Goodier represented as Ref. [15], and assumed the airy stress function, ϕ , as

$$\phi(r,\theta) = A \ln r + Br^2 \ln r + Cr^2 + D. \tag{2}$$

Since $\phi(r, \theta)$ is symmetric, it is dependent on angle only in this case. The stress fields can be yielded as

$$\sigma_{rr} = \frac{1}{r} \frac{\partial \phi}{\partial r} + \frac{1}{r^2} \frac{\partial^2 \phi}{\partial \theta^2} = \frac{A}{r^2} + B(1 + 2\ln r) + 2C,\tag{3}$$

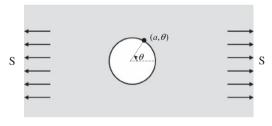


Fig. 1. An infinite plate with a circular hole subject to remote tension.

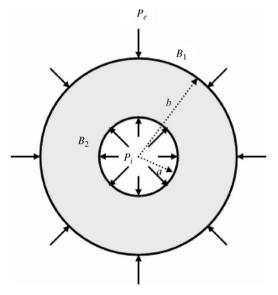


Fig. 2. An annular cylinder subject to uniform pressures.

$$\sigma_{\theta\theta} = \frac{\partial^2 \phi}{\partial r^2} = -\frac{A}{r^2} + B(3 + 2\ln r) + 2C,$$
 (4)

$$\sigma_{r\theta} = -\frac{\partial}{\partial r} \left(\frac{1}{r} \frac{\partial \phi}{\partial \theta} \right) = 0. \tag{5}$$

In order to ensure that the displacement field is a single-valued function, the coefficient B must be zero in the general solution. Eqs. (3) and (4) become

$$\sigma_{rr} = \frac{A}{r^2} + 2C,\tag{6}$$

$$\sigma_{\theta\theta} = -\frac{A}{r^2} + 2C. \tag{7}$$

Two boundary conditions $(\sigma_{rr}|_{r=b} = -P_o \text{ and } \sigma_{rr}|_{r=a} = -P_i)$ for outer and inner boundaries, respectively, are needed to be satisfied. The two coefficients, A and C, are obtained by

$$A = \frac{a^2b^2(P_o - P_i)}{b^2 - a^2},\tag{8}$$

$$C = \frac{1}{2} \frac{a^2 P_i - b^2 P_o}{b^2 - a^2}. (9)$$

After obtaining the coefficients, the Airy stress function is obtained as

$$\phi(r,\theta) = \frac{a^2b^2(P_o - P_i)}{b^2 - a^2} \ln r + \frac{1}{2} \frac{a^2P_i - b^2P_o}{b^2 - a^2} r^2 + D,$$
 (10)

where D is a constant which can be interpreted as a rigid body term. The stresses in Eqs. (3) and (4) are obtained

$$\sigma_{rr} = \frac{a^2b^2(P_o - P_i)}{b^2 - a^2} \frac{1}{r^2} + \frac{a^2P_i - b^2P_o}{b^2 - a^2},$$
(11)

$$\sigma_{\theta\theta} = -\frac{a^2b^2(P_o - P_i)}{b^2 - a^2} \frac{1}{r^2} + \frac{a^2P_i - b^2P_o}{b^2 - a^2}.$$
 (12)

For the special case of zero external pressure ($P_o = 0$), Eqs. (11) and (12) give

$$\sigma_{rr} = \frac{a^2 P_i}{b^2 - a^2} \left(1 - \frac{b^2}{r^2} \right),\tag{13}$$

$$\sigma_{\theta\theta} = \frac{a^2 P_i}{b^2 - a^2} \left(1 + \frac{b^2}{r^2} \right). \tag{14}$$

2.2.2. Stress concentration factor problem

In this case, it can be seen as the extension from the Lamé problem, when the outer radius *b* approaches infinity. In the procedure of solution, the annular case is the considered domain to analyze the problem. For the far field at infinity, the stresses are

$$\sigma_{rr}|_{r=b} = S\cos^2\theta = \frac{1}{2}S(1+\cos 2\theta),$$
 (15)

$$\sigma_{\theta\theta}|_{r=b} = S\sin^2\theta = \frac{1}{2}S(1-\cos 2\theta),\tag{16}$$

$$\sigma_{r\theta}|_{r=b} = -\frac{1}{2}S\sin 2\theta. \tag{17}$$

The traction-free boundary condition is satisfied on the boundary of the hole. The stresses on the outer boundary can be decomposed into two parts. One is the constant normal stress (S/2), and it can be calculated by using Eqs. (11) and (12). The Airy

stress function of this part is

$$\phi(r,\theta) = -\frac{S}{2}a^2 \ln r + \frac{S}{4}r^2. \tag{18}$$

The other part consists of the normal stress $(1/2)S\cos 2\theta$ and the shear stress $-(1/2)S\sin 2\theta$. The Airy stress function was assumed [15]

$$\phi(r,\theta) = \left(Ar^2 + Br^4 + C\frac{1}{r^2} + D\right)\cos 2\theta.$$
 (19)

The stress components are

$$\sigma_{rr} = \frac{1}{r} \frac{\partial \phi}{\partial r} + \frac{1}{r^2} \frac{\partial^2 \phi}{\partial \theta^2} = -\left(2A + \frac{6C}{r^4} + \frac{4D}{r^2}\right) \cos 2\theta. \tag{20}$$

$$\sigma_{\theta\theta} = \frac{\partial^2 \phi}{\partial r^2} = \left(2A + 12Br^2 + \frac{6C}{r^4}\right)\cos 2\theta \tag{21}$$

$$\sigma_{r\theta} = -\frac{\partial}{\partial r} \left(\frac{1}{r} \frac{\partial \phi}{\partial \theta} \right) = \left(2A + 6Br^2 - \frac{6C}{r^4} - \frac{2D}{r^2} \right) \sin 2\theta. \tag{22}$$

By employing Eqs. (20)–(22) to satisfy the boundary condition of Eqs. (15)–(17) and setting $a/b \approx 0$, we obtain coefficients

$$A = -\frac{S}{4}, \quad B = 0, \quad C = -\frac{a^4}{4}S, \quad D = \frac{a^2}{2}S,$$
 (23)

by using the Mathematica manipulation. The Airy stress function is obtained.

$$\phi(r,\theta) = \left(-\frac{S}{4}r^2 - \frac{a^4S}{4}\frac{1}{r^2} + \frac{a^2S}{2}\right)\cos 2\theta. \tag{24}$$

Therefore, we have the total stress function by superimposing two parts

$$\phi(r,\theta) = -\frac{a^2S}{2}\ln r + \frac{S}{4}r^2 + \left(-\frac{S}{4}r^2 - \frac{a^4S}{4}\frac{1}{r^2} + \frac{a^2S}{2}\right)\cos 2\theta.$$
 (25)

The stress components are

$$\sigma_{rr} = \frac{S}{2} \left(1 - \frac{a^2}{r^2} \right) + \frac{S}{2} \left(1 + \frac{3a^4}{r^4} - \frac{4a^2}{r^2} \right) \cos 2\theta, \tag{26}$$

$$\sigma_{\theta\theta} = \frac{S}{2} \left(1 + \frac{a^2}{r^2} \right) - \frac{S}{2} \left(1 + \frac{3a^4}{r^4} \right) \cos 2\theta, \tag{27}$$

$$\sigma_{r\theta} = -\frac{S}{2} \left(1 - \frac{3a^4}{r^4} + \frac{2a^2}{r^2} \right) \sin 2\theta. \tag{28}$$

By substituting r = a in Eq. (27), we find the hoop stress

$$\sigma_{\theta\theta} = S - 2S\cos 2\theta, \quad \sigma_{rr} = \sigma_{r\theta} = 0.$$
 (29)

It is noted that the hoop stress $(\sigma_{\theta\theta})$ reaches the maximum of 3*S* when $\theta = \pi/2$ or $\theta = 3\pi/2$.

2.3. Trefftz formulation—the present approach

In the Trefftz method, the field solution u(x) is

$$u(x) = \sum_{j=1}^{N_{T}} c_{j} u_{j}(x), \tag{30}$$

where N_T is the number of T-complete functions, c_j is the jth unknown coefficient and $u_j(x)$ is the jth T-complete function which satisfies the governing equation. The T-complete functions of biharmonic problem of the Airy stress function for interior and exterior cases can be found in Table 1 [11].

Table 1T-complete functions of the Trefftz method and degenerate kernels of the MFS for the biharmonic problem

Method of fundamental solution (MFS)		Trefftz method	
Fundamental solution	Degenerate kernel	Interior basis	Exterior basis
1D (1/12)r ³	Basis functions and degenerate kernels $U(s,x) = \begin{cases} \frac{1}{12}(x^3 - 3x^2s + 3xs^2 - s^3), & x > s \\ \frac{1}{12}(s^3 - 3s^2x + 3sx^2 - x^3), & x < s \end{cases}$	1, x, x ² , x ³	1, x, x ² , x ³
2D <i>r</i> ² ln(<i>r</i>)	$U(s,x) = \begin{cases} U^{l}(s,x) = & \rho^{2}(1+\ln R) + R^{2} \ln R - R\rho(1+2\ln R)\cos(\theta-\phi) \\ & -\sum_{m=1}^{\infty} \frac{1}{m(m+1)} \frac{\rho^{m+2}}{R^{m}} \cos[m(\theta-\phi)] \\ & +\sum_{m=2}^{\infty} \frac{1}{m(m-1)} \frac{\rho^{m}}{R^{m-2}} \cos[m(\theta-\phi)], R > \rho \end{cases}$ $U(s,x) = \begin{cases} U^{E}(s,x) = & R^{2}(1+\ln \rho) + \rho^{2} \ln \rho - \rho R(1+2\ln \rho)\cos(\theta-\phi) \\ & -\sum_{m=1}^{\infty} \frac{1}{m(m+1)} \frac{R^{m+2}}{\rho^{m}} \cos[m(\theta-\phi)] \\ & +\sum_{m=2}^{\infty} \frac{1}{m(m-1)} \frac{R^{m}}{\rho^{m-2}} \cos[m(\theta-\phi)], \rho > R \end{cases}$	1, ρ^2 , $\rho^m \cos(m\phi)$, $\rho^m \sin(m\phi)$, $\rho^{m+2} \cos(m\phi)$, $\rho^{m+2} \sin(m\phi)$	$\ln(\rho), \rho^2 \ln(\rho), \rho^{-m} \cos(m\phi), \rho^{-m} \sin(m\phi),$ $\rho^{2-m} \cos(m\phi), \rho^{2-m} \sin(m\phi)$
The basis func	tion which satisfy the equation		
1D 2D	$(\partial U^4(x, s)/\partial x^4) = \delta(x-s)$ $\nabla^4 U(x, s) = 8\pi\delta(x-s)$	$d^4 u(x)/dx^4 = 0$ $\nabla^4 u(\rho, \phi) = 0$	

2.3.1. Lamé problem

For the Lamé problem, the Airy stress function is obtained

$$\begin{split} \phi(r,\theta) &= \bar{a}_0 + \sum_{m=1}^N \bar{a}_m r^m \cos m\theta + \sum_{m=1}^N \bar{b}_m r^m \sin m\theta \\ &+ \bar{c}_0 r^2 + \sum_{m=1}^N \bar{c}_m r^{m+2} \cos m\theta + \sum_{m=1}^N \bar{d}_m r^{m+2} \sin m\theta \\ &+ a_0 \ln r + \sum_{m=1}^N a_m r^{-m} \cos m\theta + \sum_{m=1}^N b_m r^{-m} \sin m\theta \\ &+ c_0 r^2 \ln r + \sum_{m=1}^N c_m r^{2-m} \cos m\theta + \sum_{m=1}^N d_m r^{2-m} \sin m\theta, \end{split}$$

by using the Trefftz base. Since it can be seen as an interior problem superimposing with an exterior case, both interior and exterior Trefftz bases are chosen. Based on the relation between stress and Airy stress function, we have

$$\sigma_{rr} = \sum_{m=1}^{N} (m - m^{2}) \bar{a}_{m} r^{m-2} \cos m\theta + \sum_{m=1}^{N} (m - m^{2}) \bar{b}_{m} r^{m-2} \sin m\theta$$

$$+ 2\bar{c}_{0} + \sum_{m=1}^{N} (m + 2 - m^{2}) \bar{c}_{m} r^{m} \cos m\theta$$

$$+ \sum_{m=1}^{N} (m + 2 - m^{2}) \bar{d}_{m} r^{m} \sin m\theta + a_{0} \frac{1}{r^{2}}$$

$$- \sum_{m=1}^{N} (m + m^{2}) a_{m} r^{-(m+2)} \cos m\theta$$

$$- \sum_{m=1}^{N} (m + m^{2}) b_{m} r^{m-2} \sin m\theta + c_{0}(2 \ln r + 1)$$

$$+ \sum_{m=1}^{N} (2 - m - m^{2}) c_{m} r^{-m} \cos m\theta$$

$$+ \sum_{m=1}^{N} (2 - m - m^{2}) d_{m} r^{-m} \sin m\theta.$$
(32)

By matching the boundary condition $(\sigma_{rr}|_{r=b}=-P_o)$ for the outer boundary, we have

$$2\bar{c}_0 + a_0 \frac{1}{h^2} + c_0 (2 \ln b + 1) = -P_o.$$
 (33)

By matching the inner boundary condition $(\sigma_{rr}|_{r=a}=-P_i)$, we have

$$2\bar{c}_0 + a_0 \frac{1}{a^2} + c_0(2 \ln a + 1) = -P_i.$$
 (34)

Since the solution is a single-valued function, c_0 must be zero. From Eqs. (33) and (34), we have

$$a_0 = \frac{a^2b^2}{b^2 - a^2}(P_o - P_i),\tag{35}$$

$$\bar{c}_0 = \frac{1}{2} \frac{a^2 P_i - b^2 P_o}{b^2 - a^2}.$$
 (36)

Therefore, the Airy stress function is

$$\phi(r,\theta) = \frac{1}{2} \frac{a^2 P_i - b^2 P_o}{b^2 - a^2} r^2 + \frac{a^2 b^2 (P_o - P_i)}{b^2 - a^2} \ln r + \bar{a}_0, \tag{37}$$

where \bar{a}_0 is a constant. Then, the stress components are

$$\sigma_{rr} = \frac{a^2 P_i - b^2 P_o}{b^2 - a^2} + \frac{a^2 b^2 (P_o - P_i)}{b^2 - a^2} \frac{1}{r^2},\tag{38}$$

$$\sigma_{\theta\theta} = \frac{a^2 P_i - b^2 P_o}{b^2 - a^2} - \frac{a^2 b^2 (P_o - P_i)}{b^2 - a^2} \frac{1}{r^2}.$$
 (39)

The results are the same with the Timoshenko and Goodier's solution.

2.3.2. Stress concentration factor problem

For the problem of an infinite plate with a circular hole subject to a uniform tension of magnitude S in the x direction, it can be decomposed into two parts by using the superposition technique, as shown in Fig. 3(a) and (b). One is an infinite plate subject to a uniform tension and the other is an infinite plate with a free-traction hole. On the boundary of the hole, it needs to satisfy the boundary conditions of traction free for the superposing total solution. For the problem of an infinite plate subject to a uniform tension, it can be seen as a circular plate with an infinite radius. The Airy stress function is represented as

$$\phi^{\infty}(r,\theta) = \bar{a}_0 + \sum_{m=1}^{N} \bar{a}_m r^m \cos m\theta + \sum_{m=1}^{N} \bar{b}_m r^m \sin m\theta + \bar{c}_0 r^2 + \sum_{m=1}^{N} \bar{c}_m r^{m+2} \cos m\theta + \sum_{m=1}^{N} \bar{d}_m r^{m+2} \sin m\theta,$$
(40)

by choosing the interior Trefftz bases in Table 1. We have the stress components as follows:

$$\sigma_{rr}^{\infty} = \sum_{m=1}^{N} (m - m^{2}) \bar{a}_{m} r^{m-2} \cos m\theta + \sum_{m=1}^{N} (m - m^{2}) \bar{b}_{m} r^{m-2} \sin m\theta + 2\bar{c}_{0} + \sum_{m=1}^{N} (m + 2 - m^{2}) \bar{c}_{m} r^{m} \cos m\theta + \sum_{m=1}^{N} (m + 2 - m^{2}) \bar{d}_{m} r^{m} \sin m\theta,$$
 (41)

$$\sigma_{r\theta}^{\infty} = \sum_{m=1}^{N} m(m-1)\bar{a}_{m}r^{m-2} \sin m\theta - \sum_{m=1}^{N} m(m-1)\bar{b}_{m}r^{m-2} \cos m\theta + \sum_{m=1}^{N} m(m+1)\bar{c}_{m}r^{m} \sin m\theta - \sum_{m=1}^{N} m(m+1)\bar{d}_{m}r^{m} \cos m\theta,$$
(42)

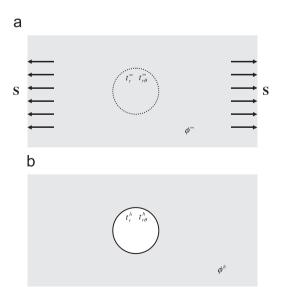


Fig. 3. (a) An infinite plate subject to a uniform tension (ϕ^{∞} field) and (b) an infinite plate with a hole (ϕ^h field).

$$\sigma_{\theta\theta}^{\infty} = \sum_{m=1}^{N} m(m-1)\bar{a}_{m}r^{m-2}\cos m\theta + \sum_{m=1}^{N} m(m-1)\bar{b}_{m}r^{m-2}\sin m\theta + 2\bar{c}_{0} + \sum_{m=1}^{N} (m+2)(m+1)\bar{c}_{m}r^{m}\cos m\theta + \sum_{m=1}^{N} (m+2)(m+1)\bar{d}_{m}r^{m}\sin m\theta.$$
(43)

When r approaches infinity, the boundary conditions, $\sigma_n^\infty = (S/2)(1+\cos 2\theta)$, $\sigma_{r\theta}^\infty = -(1/2)S\sin 2\theta)$ and $\sigma_{\theta\theta}^\infty = (S/2)(1-\cos 2\theta)$, are needed to be satisfied. After comparing with the coefficient, we have

$$\bar{a}_2 = -\frac{1}{4}S, \quad \bar{c}_0 = \frac{1}{4}S, \quad \bar{a}_0 = \text{arbitrary}$$
 (44)

and all other coefficients are equal to zero. The Airy stress function for the problem is obtained

$$\phi^{\infty}(r,\theta) = \frac{S}{4}r^2 - \frac{S}{4}r^2 \cos 2\theta + \bar{a}_0.$$
 (45)

For the other part of the problem, an infinite plate with a hole, we choose the exterior Trefftz base for the Airy stress function as

$$\phi^{h}(r,\theta) = a_0 \ln r + \sum_{m=1}^{N} a_m r^{-m} \cos m\theta + \sum_{m=1}^{N} b_m r^{-m} \sin m\theta + c_0 r^2 \ln r + \sum_{m=1}^{N} c_m r^{2-m} \cos m\theta + \sum_{m=1}^{N} d_m r^{2-m} \sin m\theta.$$
(46)

The stress components are

$$\sigma_{rr}^{h} = \frac{a_0}{r^2} - \sum_{m=1}^{N} (m^2 + m) a_m r^{-(m+2)} \cos m\theta$$

$$- \sum_{m=1}^{N} (m^2 + m) b_m r^{-(m+2)} \sin m\theta + c_0 (2 \ln r + 1)$$

$$+ \sum_{m=1}^{N} (2 - m - m^2) c_m r^{-m} \cos m\theta$$

$$+ \sum_{m=1}^{N} (2 - m - m^2) d_m r^{-m} \sin m\theta,$$
(47)

$$\sigma_{r\theta}^{h} = -\sum_{m=1}^{N} m(m+1)a_{m}r^{-(m+2)} \sin m\theta + \sum_{m=1}^{N} m(m+1)b_{m}r^{-(m+2)} \cos m\theta + \sum_{m=1}^{N} m(1-m)c_{m}r^{-m} \sin m\theta - \sum_{m=1}^{N} m(1-m)d_{m}r^{-m} \cos m\theta,$$
(48)

$$\sigma_{\theta\theta}^{h} = -\frac{a_{0}}{r^{2}} + \sum_{m=1}^{N} m(m+1)a_{m}r^{-(m+2)} \cos m\theta$$

$$+ \sum_{m=1}^{N} m(m+1)b_{m}r^{-(m+2)} \sin m\theta + c_{0}(2 \ln r + 3)$$

$$+ \sum_{m=1}^{N} (2-m)(1-m)c_{m}r^{-m} \cos m\theta$$

$$+ \sum_{m=1}^{N} d_{m}(2-m)(1-m)r^{-m} \sin m\theta.$$
(49)

We have

$$a_0 = -\frac{S}{2}a^2$$
, $a_2 = -\frac{S}{4}a^4$, $c_2 = \frac{S}{2}a^2$, $c_0 = \text{arbitrary}$. (50)

Since the solution is a single-valued function in physics, c_0 must be zero. The Airy stress function of this part is

$$\phi^{h}(r,\theta) = -\frac{S}{2}a^{2} \ln r - \frac{S}{4}\frac{a^{4}}{r^{2}}\cos 2\theta + \frac{S}{2}a^{2}\cos 2\theta.$$
 (51)

Then, the total Airy stress function is

$$\begin{split} \phi &= \phi^{\infty} + \phi^h \\ &= \frac{S}{4} (r^2 - 2a^2 \ln r) - \frac{S}{4} (r^2 + \frac{a^4}{r^2} - 2a^2) \cos 2\theta + \bar{a}_0. \end{split} \tag{52}$$

After obtaining the Airy stress function, we can obtain the corresponding stress components

$$\sigma_{rr} = \frac{S}{2} \left(1 - \frac{a^2}{r^2} \right) + \frac{S}{2} \left(1 + \frac{3a^4}{r^4} - \frac{4a^2}{r^2} \right) \cos 2\theta, \tag{53}$$

$$\sigma_{\theta\theta} = \frac{S}{2} \left(1 + \frac{a^2}{r^2} \right) - \frac{S}{2} \left(1 + \frac{3a^4}{r^4} \right) \cos 2\theta, \tag{54}$$

$$\sigma_{r\theta} = -\frac{S}{2} \left(1 - \frac{3a^4}{r^4} + \frac{2a^2}{r^2} \right) \sin 2\theta. \tag{55}$$

On the circular boundary (r = a) of the hole, the hoop stress is

$$\sigma_{\theta\theta} = S - 2S\cos 2\theta. \tag{56}$$

When θ approaches $\pi/2$ or $3\pi/2$, $\sigma_{\theta\theta}$ reaches the maximum 3S. The result is the same with the Timoshenko and Goodier's solution. Fig. 4 shows the contour of hoop stress along the hole boundary in

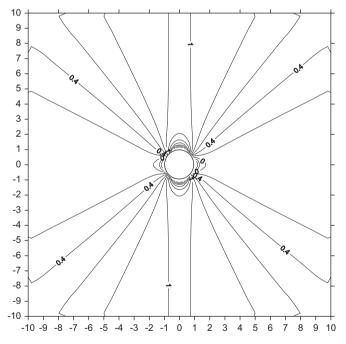


Fig. 4. The contour of hoop stress of the infinity plate.

Table 2Comparison of the present method and Timoshenko and Goodier's approach for the SCF problem.

	Present method	Timoshenko and Goodier
Coordinate	Polar coordinates	Polar coordinates
Superposition	Stress (on boundary)	Stress (at infinity)
Geometry	Interior case+exterior case	Two annular domains
Base function	T-complete function	Assumption
Method	Direct	Semi-inverse

the infinite plate. Good agreement is made. The comparison between the present method and the Timoshenko and Goodier's approach is summarized in Table 2.

3. Concluding remarks

For the two classical elasticity problems, we have revisited the analytical solution by using the Trefftz method. Instead of using inverse or semi-inverse approach in the textbook, this paper has derived the solution in a direct, logical and natural way. The stress concentration factor problem and the Lamé problem were demonstrated to see the validity of the Trefftz formulation. Good agreements were made after comparing the exact solutions with those of Timoshenko and Goodier's textbook.

Acknowledgements

Financial support from Grant No. NSC-97-2211-E-019-015-MY3 and MOE-CMBB-97-G-A-601 is gratefully acknowledged.

References

 Trefftz E. Ein Gegenstück zum ritzschen Verfahren. In: Proc second Int Cong Appl Mech. Zurich; 1926, p. 131–7.

- [2] Jin WG, Cheung YK, Zienkiewicz OC. Application of the Trefftz method in plane elasticity problems. Int J Numer Methods Eng 1990;30:1147–61.
- [3] Herrera I. Boundary methods: An algebraic theory. Boston: Pitman Advanced Pub Program; 1984.
- [4] Cheung YK, Jin WG, Zienkiewicz OC. Direct solution procedure for solution of harmonic problems using complete nonsingular Trefftz functions. Comm Appl Numer Meth 1989;5:159–69.
- [5] Cheung YK, Jin WG, Zienkiewicz OC. Solution of Helmholtz equation by Trefftz method. Int | Numer Methods Eng 1991;32:63–78.
- [6] Jirousek J, Venkatesh A. Hybrid-Trefftz plane elasticity elements with p-method capabilities. Int J Numer Methods Eng 1992;35:1443–72.
- [7] Jirousek J, Wroblewski A. T-elements: state-of-the-art and future trends. Arch Comput Methods Eng 1996;3-4:323-434.
- [8] Kita E, Kamiya N. Trefftz method: an overview. Adv Eng Software 1995; 24:3-12.
- [9] Li ZC, Lu TT, Huang HT, Cheng AHD. Trefftz, collocation, and other boundary methods—a comparison. Numer Methods PDE 2008;24(3):972–90.
- [10] Li ZC, Lu TT, Hu HY, Cheng AHD. Trefftz and collocation methods. Boston-Southampton: WIT Press; 2008.
- [11] Chen JT, Wu CS, Lee YT, Chen KH. On the equivalence of the Trefftz method and method of fundamental solutions for Laplace and biharmonic equations. Comp Math Appl 2007;53:851–79.
- [12] Schaback R. Adaptive numerical solution of MFS systems. ICCES Special Symposium on Meshless Methods. Greece: Patras; 2007.
- [13] Kaw AK, Besterfield G, Nichani S. Integrating a research problem in a course in applied elasticity. Int J Mech Eng Edu 2004;32:232–42.
- [14] Lee YT, Chen JT, Chou KS. Revisit of two classical elasticity problems by using the null-field BIE. APCOM'07 in conjunction with EPMESC XI, Japan: Kyoto; 2007.
- [15] Timoshenko SP, Goodier JN. Theory of elasticity. New York: McGraw-Hill; 1970.
- [16] Lamé G. Leçons sur la théorie de l'élasticité. Paris: Gauthier-Villars; 1852.