# A Note on the Application of Large Mass and Stiffness Techniques for Multi-support Excitations





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Why the Results Diverge by Using MSC/NASTRAN?

Rayleigh Damping Proportional to Mass Only

$$[C] = \alpha[M] + \beta[K], \ \beta = 0$$

- Large Stiffness Method Considering the Former Modes
- Discontinuity Between Boundary (B.C.) and Initial Conditions(I.C.)

$$t \to 0 \quad x \to 0$$

$$u(x,t) \neq u(x,t)$$

$$(B.C.) \quad (I.C.)$$

$$0 \quad (I.C.) \quad 1$$

Sources of Divergence in Modal Dynamics by Crandall

S. H. Crandall and A. Yildiz, Transactions of the ASME, 1962
 Beam modal (Euler-Bernoulli, Rayleigh, Timoshenko and shear beam)
 Damping mechanism (Rayleigh, viscoelastic, internal, external damping)
 Random response (primary and secondary fields)

Loading type (boundary force and support excitations are not available)

Transient response is not available

#### Sources of Divergence in This Paper

- Rayleigh damping proportional to mass only for support motion
- Series solution without considering the boundary effect for support motion
- Large stiffness technique without considering the contributions of high frequency modes
- Discontinuity between boundary and initial conditions

#### New Concept of Modal Participation Factor(MPF)

- Modal displacement governs the MPF for body force excitations
- Modal displacement governs the MPF for boundary force excitations
- Modal reaction governs the MPF for boundary support excitations

• 
$$\Gamma_{ij} = \frac{R_{ij}}{(-\omega_i^2)}$$
 support excitation force excitation

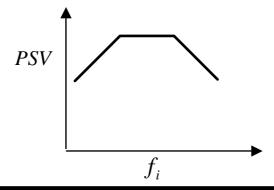
### Discrete MDOF System

### Support Motion Problems

• Response ← Superposition of Modal Response (ABS, CQC, SRSS)

—— Modal Participation Factor  $\; \Gamma_{\!_i}$ 

+ Response Spectrum



#### **Formulations**

Governing Equation:

$$M_{ll}$$
  $M_{lr}$   $M_{rr}$   $M$ 

• Decomposition:

$$U = U_{U_s} + U_{U_s} + U_{U_s} + U_{U_s}$$

• Quasi-static Part: 
$$\{U^s\} = \begin{bmatrix} K_{ll}^{-1}K_{lr} \\ I \end{bmatrix} U_r \mathbf{Q} = \begin{bmatrix} I_{G_1}\mathbf{Q} \cdots \mathbf{I}_{G_j}\mathbf{S} \cdots \mathbf{0}_{G_{N_r}} \mathbf{t} \end{bmatrix} \begin{bmatrix} U_{r1}(t) \\ \mathbf{M} \\ U_{rj}(t) \\ \mathbf{M} \end{bmatrix}$$

#### Modal Formulations

Mode superposition

$$\mathbf{M} U^d \mathbf{\Gamma} = [\Phi_{Nl}] \mathbf{M} Q^d \mathbf{\Gamma}$$

Modal Equation

$$\mathcal{Q}_{i}^{d} + 2\xi_{i}\omega_{i}\mathcal{Q}_{i}^{d} + \omega_{i}^{2}Q_{i}^{d} = -\Gamma_{ij}\mathcal{Q}_{rj}^{d}(t)$$

Modal Participation Factor (Conventional Method)

$$\Gamma_{ij} = \begin{cases} \phi_i \\ 0 \end{cases} \begin{bmatrix} M \end{bmatrix} \begin{bmatrix} M \end{bmatrix} \begin{bmatrix} \mathbf{I} G_j \\ \mathbf{S} \end{bmatrix}$$

Modal Participation Factor (Modal Reaction Method)

$$\Gamma_{ij} = \frac{R_{ij}}{(-\omega_i^2)}$$

#### Methods of Solution for Multi-Support Motions

- Large Mass Technique (1983) → Multi-support Motions(1992)
- Large Stiffness Technique (1983) → Multi-support Motions(1992)
- Cesaro Sum (1890) → Base Shear Force (1992)
- Quasi-static Decomposition (1950) → Discrete System (1975)
- Stokes' Transformation (1880) → Base Shear Force (1993)

#### Methods of Solution for Multi-Support Motions

- Mode Superposition (Series Solution)
- Large Mass Technique (Including Rigid Body Modes)
- Large Stiffness Technique (Including High Frequency Modes)
- Cesaro sum
- Quasi-static Decomposition (Mindlin)
- Stokes' Transformation

#### Capabilities of Multi-support Excitation in FEM Packages

MSC/NASTRAN

Large mass technique

Large stiffness technique

ABAQUS

Single base excitation(\*BASE MOTION), modal reaction available

• SAP90, ETABS

?

MLTDYN(NTUCE)

**Modal reaction method** 

Large Stiffness Method and Large Mass Method

Large mass Large stiffness Large mass Large stiffness

#### Large Stiffness Technique

- Large Stiffness Technique (Without Including High Frequency Modes)
   Divergence (slope, moment and shear force)
- Large Stiffness Technique (Including High Frequency Modes)
   Boundary effect (moment and shear force)
- Large Stiffness Ratio 10<sup>6</sup>
- The Additional High Frequency Modes Display the Boundary Terms

#### Large Mass Technique

- Large Mass Technique (Without Including Rigid Body Modes)
   Divergence (slope, moment and shear force)
- Large Mass Technique (Including Rigid Body Modes)
   Boundary effect (moment and shear force)
- Large Mass Ratio 10<sup>6</sup>
- The Rigid Body Modes Display the Boundary Terms

#### Results of Large Stiffness Method Without Considering High Frequency Modes

**Displacement** 

**Moment** 

**Slope** 

**Shear force** 

#### Results of Large Stiffness Method Including High Frequency Modes

**Displacement** 

Moment

**Slope** 

**Shear force** 

#### Rayleigh Damping Model

• Rayleigh Damping Proportional to Mass Only  $[C] = \alpha[M] + \beta[K], \ \beta = 0$  $\alpha, \ \beta \ fixed$   $\xi_1 = 0.05 \ fixed$ 

#### Divergence Due to Mass Proportional Damping

**Displacement** 

**Moment** 

**Slope** 

**Shear force** 

Divergence of Acceleration Profile Due to Discontinuity Between Boundary and Initial Conditions at t=0 second

**Displacement** (Initial condition)

**Velocity** (Initial condition)

**Acceleration** (Divergence)

Displacement and Acceleration Histories at the Middle Point x/l=0.5 Due to Discontinuity Between Boundary and Initial Conditions

Analytical results

Displacement

Large stiffness method (with high freq. modes)

**Displacement** 

Large stiffness method (no high freq. modes)

**Displacement** 

**Acceleration** 

Acceleration

Acceleration

Relations of Series Representation, Large Stiffness Technique, Cesaro Sum, Quasi-static Decomposition and Stokes' Transformation

#### Motivations of Quasi-static Decomposition and Stokes' Transformation

#### Quasi-static decomposition

$$u(x,t) = U(x,t) + \sum_{n=0}^{N} q_n(t) u_n(x)$$

(Physical meaning)

#### **Differentiation**

$$u'(x,t) = U'(x,t) + \sum_{n=0}^{N} q_n(t) u'_n(x)$$

#### Stokes' transformation

$$u(x,t) \cong \sum_{n=0}^{N} \overline{q}_{n}(t) u_{n}(x)$$

$$= U(x,t) + \sum_{n=0}^{N} c_{n}(t) u_{n}(x) + \sum_{n=0}^{N} \overline{q}_{n}(t) u_{n}(x)$$
Asymptotic analysis

$$u'(x,t) = \sum_{n=0}^{N} b_n(t) u'_n(x) + \sum_{n=0}^{N} \overline{q}_n(t) u'_n(x)$$
Integration
$$\omega + F \cdot P \cdot P$$

#### (Mathematical way)

$$= U'(x,t) + \sum_{n=0}^{N} c_n(t) u'_n(x) + \sum_{n=0}^{N} \overline{q}_n(t) u'_n(x) - \infty + F.P.$$

$$q_n(t) = c_n(t) + \overline{q}_n(t)$$

$$U(x,t) \cong -\sum_{n=0}^{N} c_n(t) u_n(x)$$

$$\sum_{n=0}^{N} b_n(t) u_n(x) =$$
 Series representation for distribution on boundary

## Three Analytical Ways and Two Simulation Techniques to Introduce the Quasi-static Part

- By Solving Boundary Value Problem Directly
   Quasi-static decomposition method (Mindlin and Goodman)
- By Integrating the Secondary Field Derived from Stokes' Transformation
   Boundary terms are available
- By Adding and Subtracting Technique Using Asymptotic Analysis

  Series representation (Eringen and Suhubi, Yeh and Liaw)
  - Large Mass Technique (MSC/NASTRAN) --- Rigid Body Modes
  - Large Stiffness Technique (MSC/NASTRAN) --- High Frequency Modes

#### Cesaro Regularization Technique

#### Series Solution(Partial Sum)

$$\begin{split} s_0 &= a_0 \\ s_1 &= a_0 + a_1 \\ s_2 &= a_0 + a_1 + a_2 \\ \mathbf{M} &= \mathbf{M} \\ s_{n-1} &= a_0 + a_1 + a_2 + \mathbf{L} + a_{n-1} \\ (\textit{partial sum}) \quad s_n &= a_0 + a_1 + a_2 + \mathbf{L} + a_{n-1} + a_n \quad (\textit{divergent}, \quad n \to \infty) \\ \frac{s_0 + s_1 + \mathbf{L} + s_{n-1} + s_n}{n+1} &= a_0 + \frac{n}{n+1} a_1 + \frac{n-1}{n+1} a_2 + \mathbf{L} + \frac{2}{n+1} a_{n-1} + \frac{1}{n+1} a_n \quad (\textit{convergent}, \quad n \to \infty) \\ (\textit{Cesaro sum}) \quad S_n &= \frac{1}{n+1} \sum_{k=0}^n \quad (n-k+1) \quad a_k \quad (\textit{moving average}) \end{split}$$

#### Stokes' Transformation

- Term by Term Differentiation Is Not Always Legal
- Boundary Term Is Present for Some Cases

$$f'(x) = \frac{d}{dx} f(x) = \frac{d}{dx} \sum_{k=0}^{n} c_k u_k(x) = \sum_{k=0}^{n} c_k u_k'(x) + \sum_{k=0}^{n} b_k u_k'(x)$$

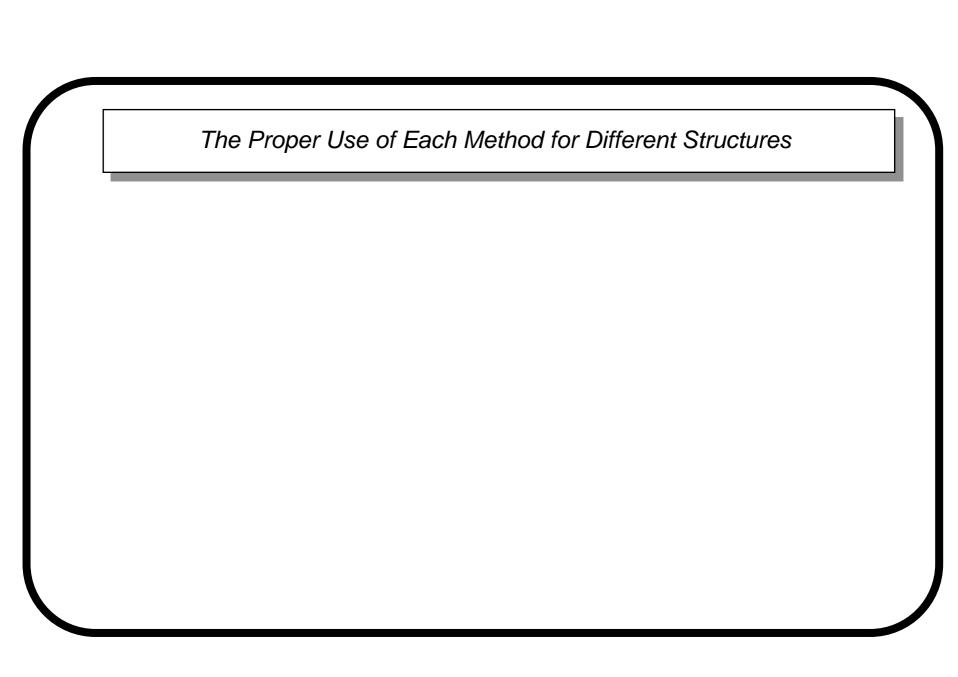
if 
$$\sum_{k=0}^{n} b_k u_k(x) \neq 0$$

**Boundary term** 

Term by Term Differentiation Is Legal

if 
$$\sum_{k=0}^{n} b_k u_k(x) = 0$$





#### Conclusions

- Sources of Divergence Have Been Identified
- New Point of View for Modal Participation Factor has been developed Physical meaning

To save CPU time (100 : 1)

- New Method for Multi-support Motion --- Stokes' Transformation
   Free from calculating quasi-static solution
   Accelerate convergence rate
- Large Stiffness and Large Mass Technique Have Been Tested
   Boundary effect of moment and shear diagrams should be noted